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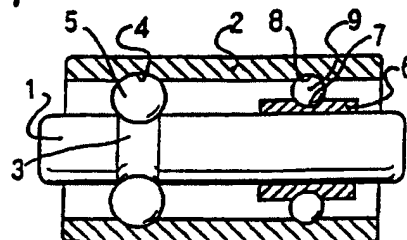
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(54) Double-row ball bearing

(57) A double-row ball bearing suitable for office automation equipment comprises an axle (1) supported for rotation within a sleeve (2) by means of first and second sets of ball bearings (5, 9). The first set of ball bearings (5) runs between an axle ball race (3) formed directly in an outer surface of the axle (1) and an outer ball race (4) which may be formed directly in an inner surface of the sleeve (2) or in an outer ring mounted in the sleeve (2). The second set of ball bearings (9) runs between an inner ring (6) mounted on the axle (1) and an outer ball race (8) formed in an inner surface of the sleeve (2). The inner ring (6) is pre-loaded and then fixed to the axle (1) to provide a suitable axial clearance between the sets of ball bearings.

FIG. 1



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Description

The present invention relates to a double-row ball bearing which is particularly adaptable to Office Automation instruments.

In ball bearings, there is a so-called "direct type of ball bearing" in which a ball race is directly formed in an outer peripheral surface of an axle.

In such direct type of ball bearing, in case that the ball bearing has a single row of balls only, it is very easy to have the balls aligned with each of an axle ball race (which is formed in an outer peripheral surface of the axle) and an outer-ring ball race (which is formed in an inner peripheral surface of an outer ring). In contrast with this, in case that the ball bearing has a pair of rows of balls, i.e., in case that the ball bearing is of a double-row type, it is very difficult to have the balls and the races aligned with the races.

This will be described with reference to Fig. 12 which shows a conventional double-row ball bearing in which: a pair of axle ball races 101a, 101b are formed in an outer peripheral surface of an axle 100 so as to extend circumferentially; a pair of outer-ring ball races 103a, 103b are formed in an inner surface of an outer ring 102; a plurality of balls 104a and 104b are disposed between the axle ball race 101a and the outer-ring ball race 103a, and between the axle ball race 101b and the outer-ring ball race 103b, respectively.

In the double-row ball bearing having the above construction, the following equation must be satisfied:

$$B > A$$

where: "A" denotes a distance between center lines of the axle ball races 101a, 101b; and "B" denotes a distance between center lines of the outer-ring ball races 103a, 103b.

In case that "B" is too larger than "A", excessive loads are applied to the balls and the races to deform the same so that the ball bearing is damaged.

Particularly, in case of miniature ball bearings, a difference between "B" and "A", and more particularly an axial clearance denoted by the reference number "C" having an amount of $(B - A) \times 1/2$ must be of the order of submicrons.

In the conventional double-row ball bearing, the pair of outer-ring ball races 103a and 103b are previously formed in the inner peripheral surface of the outer ring 102 so as to correspond to the pair of axle ball races 101a and 101b of the axle 1, respectively.

Consequently, such outer-ring ball races 103a, 103b must be formed so as to precisely correspond to the axle ball races 101a, 101b. Further, in assembling of the outer ring 102 with the axle 100, it is necessary to keep the axial clearance C at a proper value, which requires the assembling work to be conducted in an extremely precise manner. In addition, once the bearing is assembled, the thus assembled bearing can not be adjusted later.

It is an object of the present invention to provide a double-row ball bearing which: has a pre-load easily applied to an inner ring (which is mounted on an axle) or to an outer ring (which is mounted in a sleeve); enables the axial clearance (which is defined between the ball races) to be precisely set; is excellent in rigidity; improved in so-called "raceway run-out with side"; is excellent in properties resistant to vibration; is easily assembled, which reduces manufacturing costs of the double-row ball bearings of the present invention.

The above object of the present invention is accomplished by providing:

A double-row ball bearing comprising an axle and a sleeve, wherein:

a pre-load is applied to one of an inner ring (which is mounted on the axle) and an outer ring (which is mounted on a sleeve), so that an axial clearance is set at a predetermined value; and

the inner ring is fixed to the axle or the outer ring is fixed to the sleeve, in a condition in which the axial clearance is set at the predetermined value.

The above object of the present invention is accomplished by providing:

A double-row ball bearing comprising an axle and a sleeve, wherein:

an axle ball race is formed in an outer peripheral surface of the axle, while a sleeve ball race is formed an inner peripheral surface of the sleeve, so that a plurality of balls are disposed between the axle ball race and the sleeve ball race;

an inner ring, which has an inner-ring ball race formed in its outer peripheral surface, is slidably mounted on the axle, while another sleeve ball race is formed in the inner peripheral surface of the sleeve, so that a plurality of balls are disposed between the inner-ring ball race and the another sleeve ball race;

a pre-load is axially applied to the inner ring to have an axial clearance set at a predetermined value; and

the inner ring is fixed to the axle in a condition in which the axial clearance is set in the predetermined value.

The above object of the present invention is accomplished by providing:

A double-row ball bearing comprising an axle and a sleeve, wherein:

an axle ball race is formed in an outer peripheral surface of the axle;

a sleeve ball race is formed in an inner peripheral surface of the sleeve so that a plurality of balls are disposed between the axle ball race and the sleeve ball race;

another axle ball race is formed in the outer peripheral surface of the axle, while an outer-ring ball race formed in an inner peripheral surface of an outer ring which is slidably mounted in the sleeve, so that a plurality of balls are disposed between the another axle ball race and the outer-ring ball race;

a pre-load is axially applied to the outer ring so that an axial clearance is set at a predetermined value; and

the outer ring is fixed to the sleeve in a condition in which the axial clearance is set at the predetermined value.

The above object of the present invention is accomplished by providing:

A double-row ball bearing comprising an axle and a sleeve, wherein:

an axle ball race is formed in an outer peripheral surface of the axle;

a sleeve ball race is formed in an inner peripheral surface of the sleeve, so that a plurality of balls are disposed between the axle ball race and the sleeve ball race;

a ball bearing (which comprises an inner ring, an outer ring and a plurality of balls disposed between: an inner-ring ball race formed in the inner ring; and an outer-ring ball race formed in the outer ring) has its inner ring mounted on the axle and has its outer ring mounted in the sleeve;

a pre-load is axially applied to one of the inner ring and the outer ring, so that an axial clearance is set at a predetermined value; and in a condition in which the axial clearance is set at the predetermined value, the inner ring is fixed to the axle or the outer ring is fixed to the sleeve.

The above object of the present invention is accomplished by providing:

A double-row ball bearing having a construction in which:

an axle ball race is formed in an outer peripheral surface of an axle, while an inner ring is slidably or unslidably mounted on the axle; an outer-ring ball race is formed in an inner peripheral surface of an outer ring which is slidably or unslidably mounted in a sleeve, so that

a plurality of balls are disposed between the axle ball race and the outer-ring ball race;

an inner-ring ball race is formed in an outer peripheral surface of the inner ring, while a sleeve ball race is formed in an inner peripheral surface of the sleeve, so that a plurality of balls are disposed between the inner-ring ball race and the sleeve ball race;

a pre-load is axially applied to at least one of the inner ring and the outer ring so that an axial clearance is set to a predetermined value; and in a condition in which the axial clearance is set at the predetermined value, the inner ring is fixed to the axle or/and the outer ring is fixed to the sleeve.

The above object of the present invention is accomplished by providing:

A double-row ball bearing having a construction in which:

an axle ball race is formed in an outer peripheral surface of an axle, while an inner-ring ball race is formed in an outer peripheral surface of an inner ring;

a sleeve ball race is formed in an inner peripheral surface of each of a pair of sleeves (each of which serves as an outer ring, and is so provided as to correspond in position to each of the axle and the inner ring);

a plurality of balls are disposed between the axle ball race and one of the sleeve ball races, and between the inner-ring ball race and the other of the sleeve ball races;

a pre-load is axially applied to each of the sleeves so that an axial clearance is set at a predetermined value.

The above object of the present invention is accomplished by providing:

A double-row ball bearing comprising an axle and a sleeve, wherein:

an inner-ring ball race is formed in an outer peripheral surface of each of inner rings which are mounted on the axle, and at least one of the inner rings is axially slidable relative to the axle;

a pair of sleeve ball races are formed in an inner peripheral surface of the sleeve;

a plurality of balls are disposed between one of the inner-ring ball races and one of the sleeve ball races, and between the other of the inner-ring ball races and the other of the sleeve ball races;

a pre-load is axially applied to the axially slidable one of the inner rings so that an axial clear-

ance is set at a predetermined value; and the axially slidable one of the inner rings is fixed to the axle in a condition in which the axial clearance is set at the predetermined value.

In the accompanying drawings:-

Fig. 1 is a longitudinal sectional view of a first embodiment of the double-row ball bearing of the present invention;

Fig. 2 is a longitudinal sectional view of a second embodiment of the double-row ball bearing of the present invention;

Fig. 3 is a longitudinal sectional view of a third embodiment of the double-row ball bearing of the present invention;

Fig. 4 is a longitudinal sectional view of a fourth embodiment of the double-row ball bearing of the present invention;

Fig. 5 is a longitudinal sectional view of a fifth embodiment of the double-row ball bearing of the present invention;

Fig. 6 is a longitudinal sectional view of a sixth embodiment of the double-row ball bearing of the present invention;

Fig. 7 is a longitudinal sectional view of a seventh embodiment of the double-row ball bearing of the present invention;

Fig. 8 is a longitudinal sectional view of an eighth embodiment of the double-row ball bearing of the present invention;

Fig. 9 is a longitudinal sectional view of a ninth embodiment of the double-row ball bearing of the present invention;

Fig. 10 is a longitudinal sectional view of a tenth embodiment of the double-row ball bearing of the present invention;

Fig. 11 is a longitudinal sectional view of an eleventh embodiment of the double-row ball bearing of the present invention; and

Fig. 12 is a longitudinal sectional view of the conventional double-row ball bearing.

All the embodiments of the present invention are double-row ball bearings each of which is provided with an axle and a sleeve which forms an outer ring corresponding to the axle.

In a first embodiment of a double-row ball bearing of the present invention shown in Fig. 1, an axle ball race 3 and an outer-ring ball race 4 are formed in an outer peripheral surface of the axle 1 and an inner peripheral surface of the sleeve 2, respectively. A plurality of balls 5 are disposed between these ball races 3 and 4. An inner ring 6 is fixedly mounted on the axle 1. An inner-ring ball race 7 and another outer-ring ball race 8 are formed in an outer peripheral surface of the inner ring 6 and the inner peripheral surface of the sleeve 2, respectively. A plurality of balls 9 are disposed between these ball races 7 and 8.

In manufacturing of the double-row ball bearing having the above construction, the inner ring 6 is slidably mounted on the axle 1 so as to be axially movable along the axle 1. Then, the thus mounted inner ring 6 is subjected to a pre-load applied to the right face of the inner ring 6 as viewed in Fig. 1, so that the inner ring 6 is slightly moved to have the axial clearance C (which is already described above) reach a predetermined value. After the axial clearance reaches such predetermined value, the inner ring 6 is fixed to the axle 1 with a suitable adhesive agent and the like.

A method for applying the pre-load to the inner ring 6 of the double-row ball bearing of the present invention can be any one of the following methods: one using a weight; one using a spring; one using shims; one using spacers; and, one using screws.

In a second embodiment of the double-row ball bearing of the present invention shown in Fig. 2, the axle 1 is of a shoulder type provided with a small-diameter axle portion 1a on which the inner ring 6 is fixedly mounted. The remaining components of the double-row ball bearing shown in Fig. 2 are the same in construction as those of the double-row ball bearing shown in Fig. 1. In manufacturing of the double-row ball bearing shown in Fig. 2, the pre-load is applied to the inner ring 6.

Figs. 3 and 4 show a third and a fourth embodiment of the double-row ball bearings of the present invention, respectively. In each of these embodiments: an outer ring 10 forming the above-mentioned another outer-ring ball race is mounted on an inner surface of the sleeve 2. The balls 9 are disposed between: an outer-ring ball race 11 formed in an inner peripheral surface of the outer ring 10; and another axle ball race 12 formed in an outer peripheral surface of the axle 1. In the double-row ball bearing shown in Fig. 3, the sleeve 2 keeps its inner diameter constant over the full length thereof. In contrast with this, in the double-row ball bearing shown in Fig. 4, the sleeve 2 is of a shoulder type provided with a large-diameter portion 2a. The remaining components of the bearing shown in Fig. 4 are the same in construction as those of the bearing shown in Fig. 3.

In manufacturing of each of these double-row ball bearings shown in Figs. 3 and 4, the outer ring 10 is slidably mounted on the axle 1. Then, a pre-load is applied to a right face of the thus mounted outer ring 10 as viewed in each of Figs. 3 and 4 so that the axial clearance is adjusted to a predetermined value. After that, the outer ring 10 is fixed to the axle 1 with a suitable adhesive agent and the like.

In each of a fifth and a sixth embodiment of the double-row ball bearings of the present invention shown in Figs. 5 and 6, respectively: the axle 1 is of a shoulder type provided with a small-diameter portion 1a; the sleeve 2 is of the shoulder type provided with the large-diameter portion 2a; and, a ball bearing having the balls is interposed between the small-diameter portion 1a of the axle 1 and the large-diameter portion 2a of the sleeve 2.

In manufacturing of the double-row ball bearing shown in Fig. 5, the outer ring 10 is slidably mounted in the sleeve 2. Then, a pre-load is applied to the thus mounted outer ring 10. Under such circumstances, the outer ring 10 is fixed to the sleeve 2 with a suitable adhesive agent and the like.

In manufacturing of the double-row ball bearing shown in Fig. 6, the inner ring 6 is slidably mounted on the small-diameter portion 1a of the axle 1. Then, a pre-load is applied to the thus mounted inner ring 6. After that, the inner ring 6 is fixed to the small-diameter portion 1a of the axle 1 with a suitable adhesive agent.

Figs. 7 and 8 show a seventh and an eighth embodiment of the double-row ball bearings of the present invention, respectively. Each of these embodiments of the present invention has the outer ring 13 disposed in one of its opposite sides and the inner ring 6 disposed in the other side. These double-row ball bearings shown in Figs. 7 and 8 differ from each other in that: the former uses the sleeve 2 of shoulder type provided with the large-diameter portion 2a and the axle 1 of shoulder type provided with the small-diameter portion 1a; while the latter uses the sleeve 2 of straight type and the axle 1 of straight type.

In manufacturing of each of these double-row ball bearings shown in Figs. 7 and 8: both of the inner ring 6 and the outer ring 13 are slidably mounted on the axle 1 and in the sleeve 2, respectively; then, a pre-load is applied to each of the thus mounted inner ring 6 and outer ring 13; and, thereafter, the inner ring 6 and the outer ring 13 are fixed to the axle 1 and the sleeve 2 with a suitable adhesive agent, respectively.

Figs. 9 and 10 show a ninth and a tenth embodiment of the double-row ball bearings of the present invention, respectively. In each of these embodiments, the sleeve 2 serves as an outer ring.

In each of these double-row ball bearings shown in Figs. 9 and 10: a pair of the inner rings 6a, 6b are slidably mounted on the axle 1; inner-ring ball races 7a and 7b are formed in outer peripheral surfaces of the inner rings 6a and 6b, respectively; a pair of sleeve ball races 4 and 8 are formed in the inner peripheral surface of the sleeve 2 to correspond respectively in position to the inner-ring ball races 7a and 7b; and, balls 5 and 9 are disposed between the thus corresponding races 4 and 7a, and between the thus corresponding races 8 and 7b, respectively. Thereafter, a pre-load is applied to each of the inner rings 6a and 6b both of which are then fixed to the axle 1 with a suitable adhesive agent.

Fig. 11 shows an eleventh embodiment of the double-row ball bearing of the present invention. In this embodiment: a pair of axle ball races 3 and 12 are formed in an outer peripheral surface of the axle 1 so as to be axially space apart from each other, and correspond to outer-ring ball races 4 and 8, respectively, which races 4 and 8 are formed in inner peripheral surfaces of sleeves 2a and 2b, respectively; and, balls 5 and 9 are interposed between the races 3, 4 and the races 12, 8, respectively. In assembling of the double-

row ball bearing shown in Fig. 11, these sleeves 2a, 2b are mounted on the axle 1 in a condition in which a pre-load is applied to each of the sleeves 2a, 2b.

In manufacturing of the double-row ball bearing of the present invention, since one of the inner ring (which is mounted on the axle) and the outer ring (which is mounted on the sleeve) is axially slidable, it is possible to easily apply a pre-load to such axially slidable one so that the axial clearance reaches a predetermined value. Consequently, it is possible to provide the double-row ball bearings each of which is: excellent in rigidity; improved in so-called "raceway run-out with side"; and, excellent in properties resistant to vibration. In addition, each of the double-row ball bearings of the present invention is easily assembled, which reduces manufacturing costs of the double-row ball bearings of the present invention.

Claims

1. A double-row ball bearing comprising:

an axle (1);
a sleeve (2) surrounding said axle (1);
first (5) and second (9) sets of balls;
an inner ring (6) in a surface of which an inner ball race (7) for second set of balls (9) is formed;
wherein said first set of balls (5) run between an axle ball race (3) formed in an outer surface of said axle and an outer ball race (4, 11) associated with said sleeve (2);
a pre-load is applied to said inner ring (6) which is then fixed so that said first and second set of balls are set at a suitable axial position;
said axle ball race (3) is directly formed in the outer surface of said axle (1);
said pre-loaded inner ring (6) is mounted on said axle (1) and has said inner ball race (7) for said second set of balls (9) formed in its outer surface; and
said second set of balls (9) run between said inner ball race (7) of said inner ring (6) and an outer ball race (8) directly formed in an inner surface of said sleeve (2).

2. The double-row ball bearing claimed in claim 1 wherein said outer ball race (4) for said first set of balls (5) is directly formed in the inner surface of said sleeve (2).

3. The double-row ball bearing claimed in claim 1 wherein said outer ball race (11) for said first set of balls (5) is formed in an inner surface of an outer ring (13) mounted in said sleeve (2).

FIG. 1

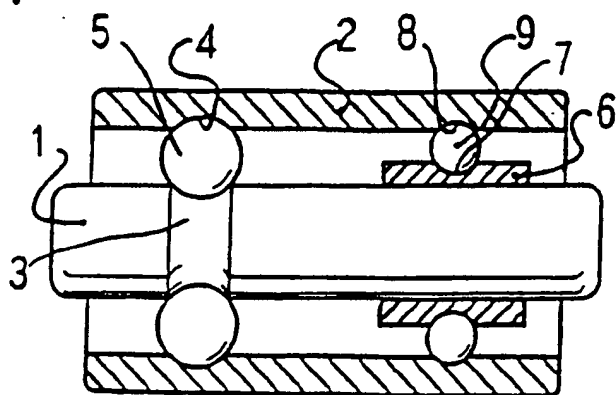


FIG. 2

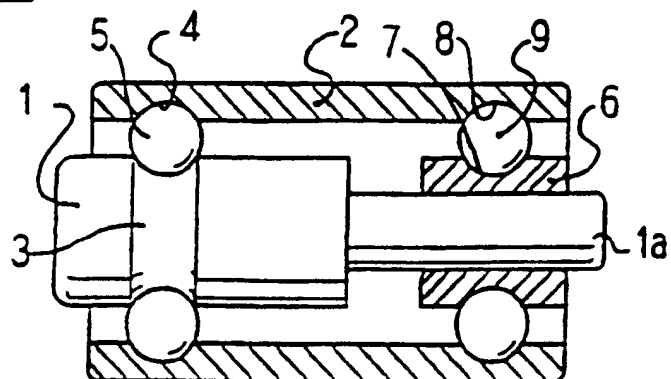


FIG. 3

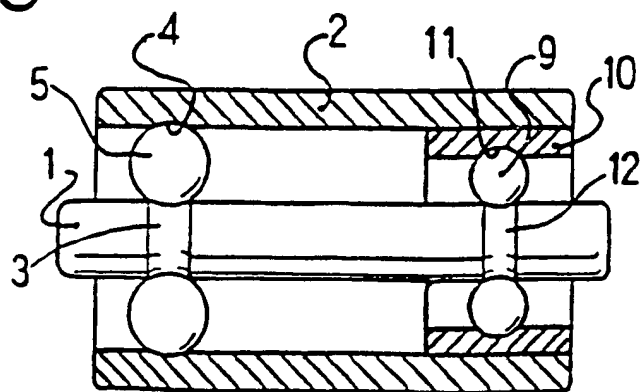


FIG. 4

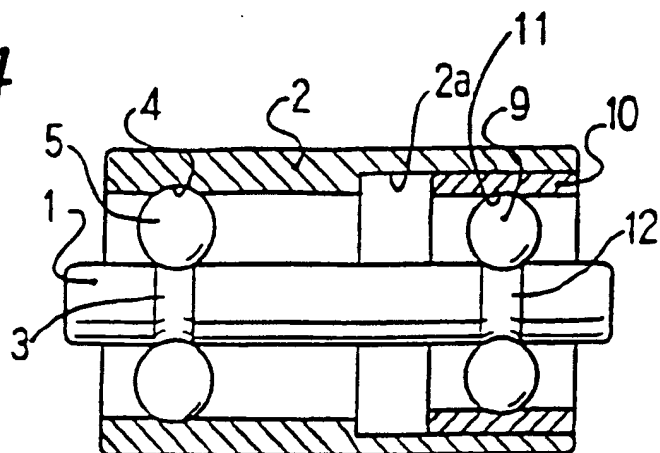


FIG. 5

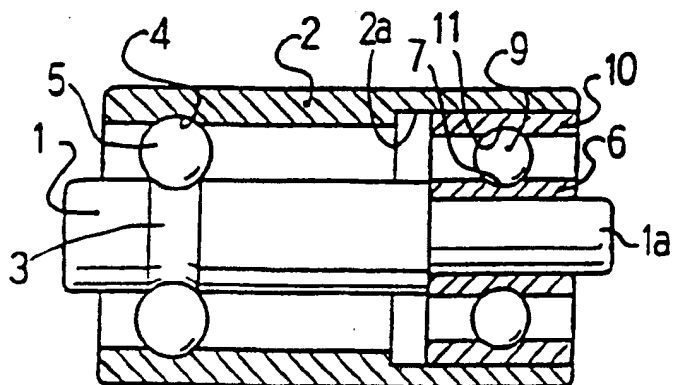


FIG. 6

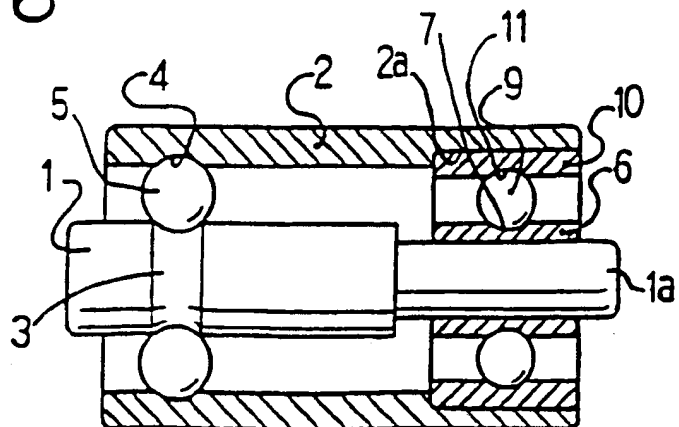


FIG. 7

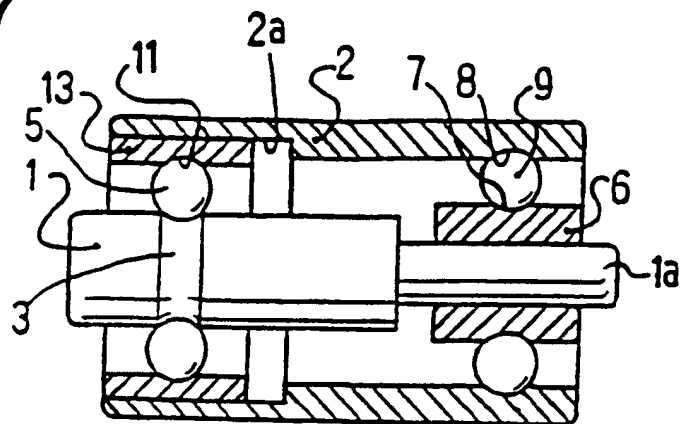


FIG. 8

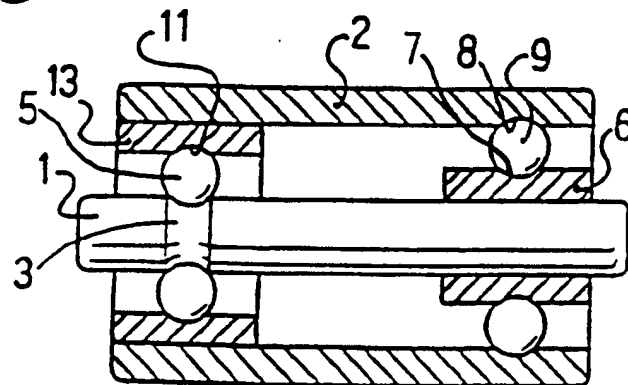


FIG. 9

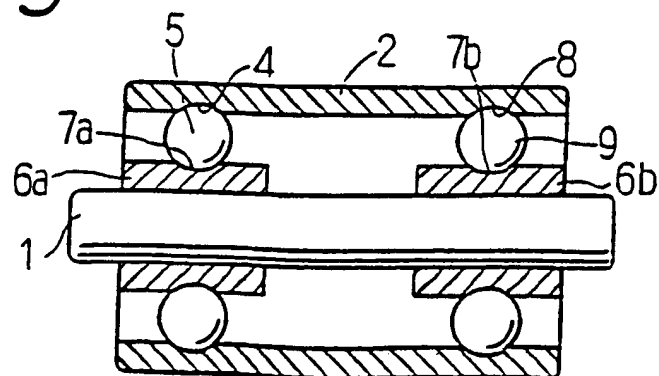


FIG. 10

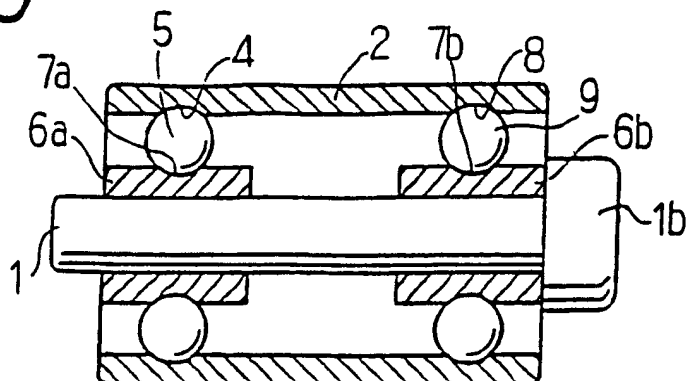


FIG. 11

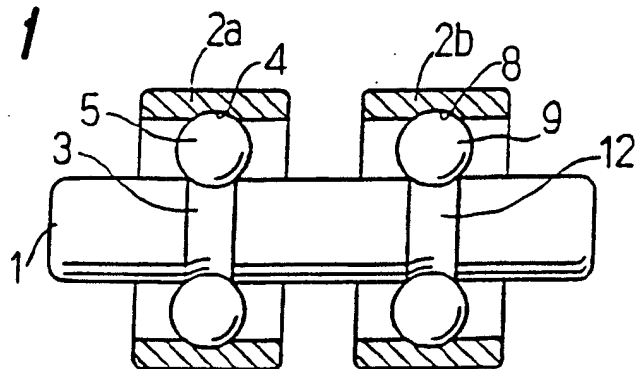
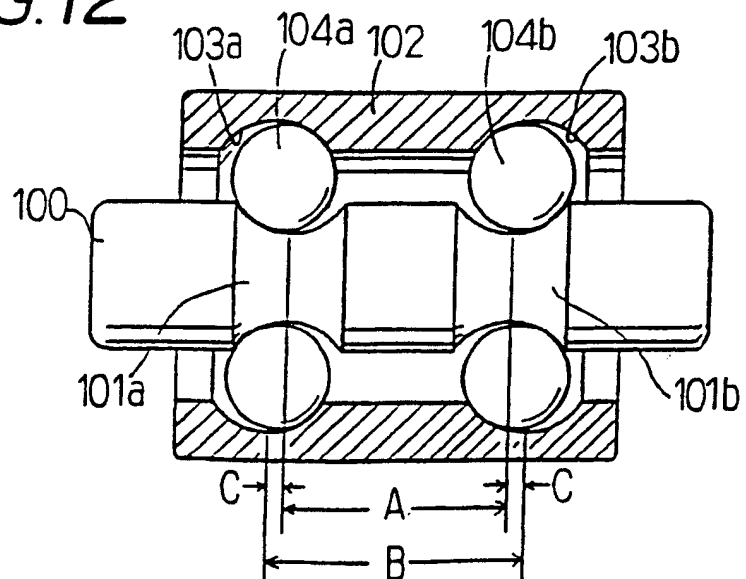
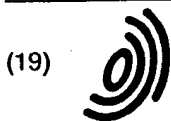


FIG. 12





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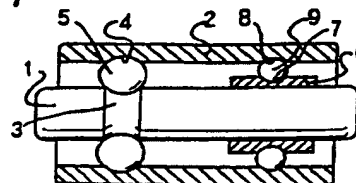
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(54) **Double-row ball bearing**

(57) A double-row ball bearing suitable for office automation equipment comprises an axle (1) supported for rotation within a sleeve (2) by means of first and second sets of ball bearings (5, 9). The first set of ball bearings (5) runs between an axle ball race (3) formed directly in an outer surface of the axle (1) and an outer ball race (4) which may be formed directly in an inner surface of the sleeve (2) or in an outer ring mounted in the sleeve (2). The second set of ball bearings (9) runs between an inner ring (6) mounted on the axle (1) and an outer ball race (8) formed in an inner surface of the sleeve (2). The inner ring (6) is pre-loaded and then fixed to the axle (1) to provide a suitable axial clearance between the sets of ball bearings.

FIG. 1



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European Patent
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EUROPEAN SEARCH REPORT

Application Number
EP 97 10 6069

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int.Cl.6)
X	US 4 713 704 A (VOLL) * column 3, line 24 - line 28 * ---	1-3	F16C19/18 F16C25/06
X	DE 31 01 596 A (SKF) * page 3 * * page 8, paragraph 5 * ---	1-3	
X	DE 31 48 191 A (SKF) * the whole document * ---	1,2	
E	US 5 206 993 A (BROUGH) * the whole document * ---	1,2	
E	US 5 226 737 A (SANDY) * the whole document * -----	1,2	
			TECHNICAL FIELDS SEARCHED (Int.Cl.6)
			F16C
The present search report has been drawn up for all claims			
Place of search THE HAGUE		Date of completion of the search 9 June 1997	Examiner Orthlieb, C
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